

The return to equilibrium problem for axisymmetric floating structures in shallow water

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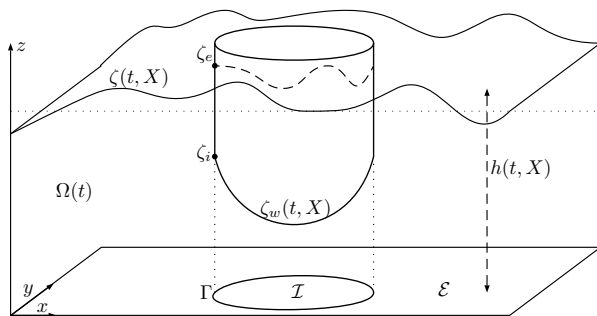
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Hydrodynamics of Wave Energy Converters 2 - IMB

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Assumptions on the solid:

- Vertical side-walls
- Only vertical motion

The contact line Γ does not depend on time

⇒ One free boundary problem: surface elevation $\zeta(t, X)$

Equations in the fluid domain $\Omega(t)$ for \mathbf{U} :

$$\partial_t \mathbf{U} + \mathbf{U} \cdot \nabla_{X,z} \mathbf{U} = -\frac{1}{\rho} \nabla_{X,z} P - g \mathbf{e}_z \quad \text{in } \Omega_t \quad (1)$$

$$\operatorname{div} \mathbf{U} = 0 \quad (2)$$

$$\operatorname{curl} \mathbf{U} = 0 \quad (3)$$

Boundary conditions at the surface and the bottom:

$$z = \zeta, \quad \partial_t \zeta - \mathbf{U} \cdot \mathbf{N} = 0 \quad \text{with } \mathbf{N} = \begin{pmatrix} -\nabla \zeta \\ 1 \end{pmatrix} \quad (4)$$

$$z = -h_0, \quad \mathbf{U} \cdot \mathbf{e}_z = 0 \quad (5)$$

Pressure in \mathcal{E} :

$$\underline{P}_e = P_{atm} \quad (6)$$

Constraint in \mathcal{I} :

$$\zeta_i(t, X) = \zeta_w(t, X) \quad (7)$$

Jump at Γ :

$$\zeta_e(t, \cdot) \neq \zeta_i(t, \cdot) \quad (8)$$

$$\underline{P}_i(t, \cdot) = P_{atm} + \rho g (\zeta_e - \zeta_i) + P_{NH} \quad (9)$$

Continuity of the normal velocity at the vertical walls:

$$\mathbf{V} \cdot \boldsymbol{\nu} = V_C \cdot \boldsymbol{\nu} \quad (10)$$

Shallow water approximation

Regime: the wavelength L is larger than the depth h_0 , i.e. $\mu = \frac{h_0^2}{L^2} \ll 1$

Nonlinear shallow water equations

At precision $O(\mu)$, h and $Q = \int_{-h_0}^{\zeta} V dz$ solve

$$\begin{cases} \partial_t h_e + \nabla \cdot Q_e = 0 \\ \partial_t Q_e + \nabla \cdot \left(\frac{1}{h_e} Q_e \otimes Q_e \right) + g h_e \nabla h_e = -\frac{h_e}{\rho} \nabla \underline{P}_e = 0 \\ \underline{P}_e = P_{atm} \end{cases} \quad \text{in } \mathcal{E}$$

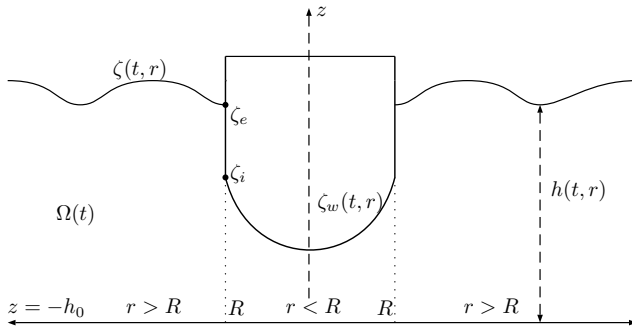
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B.C. at Γ : $\underline{P}_i|_{\Gamma} = P_{atm} + \rho g (\zeta_e - \zeta_i)|_{\Gamma} + P_{cor}$, $Q_e \cdot \nu|_{\Gamma} = Q_i \cdot \nu|_{\Gamma}$.

Axisymmetric case

Cylindrical coordinates:

$$\mathbf{U} = \mathbf{U}(t, r, \theta, z), \quad \mathbf{U} = (u_r, u_\theta, u_z) \implies Q = Q(t, r, \theta), \quad Q = (q_r, q_\theta)$$

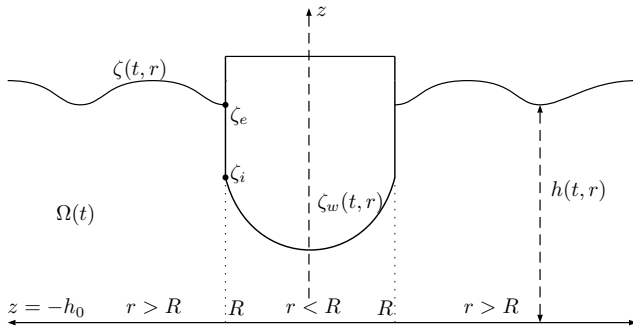


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$$\implies \mathbf{Q}(t, r) = (q_r, 0)$$

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$$\begin{cases} \partial_t h_e + \nabla \cdot Q_e = 0 \\ \partial_t Q_e + \nabla \cdot \left(\frac{1}{h_e} Q_e \otimes Q_e \right) + gh_e \nabla h_e = -\frac{h_e}{\rho} \nabla \underline{P}_e = 0 \\ \underline{P}_e = P_{atm} \end{cases} \quad \text{in } \mathcal{E}$$

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$$\text{B.C.} \quad \begin{cases} \underline{P}_i|_{\Gamma} = P_{atm} + \rho g (\zeta_e - \zeta_i)|_{\Gamma} + P_{cor}, \\ Q_e \cdot \nu|_{\Gamma} = Q_i \cdot \nu|_{\Gamma}. \end{cases} \quad \text{at } \Gamma$$

↪ Pressure eq: $-\nabla \cdot \left(\frac{h_w}{\rho} \nabla \underline{P}_i \right) = -\partial_t^2 h_w + \dots$

Axisymmetric nonlinear shallow water equations

$$\begin{cases} \partial_t h_e + \partial_r q_e + \frac{q_e}{r} = 0, \\ \partial_t q_e + \partial_r \left(\frac{q_e^2}{h_e} \right) + \frac{q_e^2}{r h_e} + g h_e \partial_r h_e = 0, \\ \underline{P}_e = P_{atm} \end{cases} \quad \text{in } (R, +\infty)$$

$$\begin{cases} \partial_t h_i + \partial_r q_i + \frac{q_i}{r} = 0, \\ \partial_t q_i + \partial_r \left(\frac{q_i^2}{h_i} \right) + \frac{q_i^2}{r h_i} + g h_i \partial_r h_i = -\frac{h_i}{\rho} \partial_r \underline{P}_i, \\ h_i = h_w \end{cases} \quad \text{in } (0, R)$$

$$\text{B.C.} \quad \begin{cases} \underline{P}_i = P_{atm} + \rho g (\zeta_e - \zeta_i) + P_{cor}, \\ q_e = q_i. \end{cases} \quad \text{at } r = R$$

$\rightsquigarrow P_{cor} \sim q_i^2|_{r=R} \left(\frac{1}{h_e^2} - \frac{1}{h_i^2} \right) |_{r=R}$ makes the system **conservative**.

Solid motion

Solid: $G(t) = (0, 0, z_G(t))$, $\mathbf{U}_G(t) = (0, 0, \dot{z}_G(t))$, $\omega = 0$

Define the displacement $\delta_G(t) := z_G(t) - z_{G,eq}$

From the assumptions on the solid: $h_w(t, r) = h_{w,eq}(r) + \delta_G(t)$

By the interior constraint $h_w = h_i$ we have also

$$q_i(t, r) = -\frac{r}{2}\dot{\delta}_G(t)$$

Newton's law for the conservation of the linear momentum

$$m\ddot{\delta}_G = -mg + \int_0^R (\underline{P}_i - P_{atm})$$

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$$m\ddot{\delta}_G = -mg + \int_0^R (\underline{P}_i - P_{atm})$$

Using the elliptic equation on \underline{P}_i

$$(m + m_a(\delta_G))\ddot{\delta}_G(t) = -c\delta_G(t) + c\zeta_e(t, R) + \left(\frac{\mathbf{b}}{h_e^2(t, R)} + \beta(\delta_G) \right) \delta_G^2(t)$$

Writing $u = (h_e, q_e)$:

Fluid part (Hyperbolic IBVP)

$$\begin{cases} \partial_t u + A(u) \partial_r u + B(u, r) u = 0, & r \in (R, +\infty) \\ \mathbf{e}_2 \cdot u|_{r=R} = -\frac{R}{2} \dot{\delta}_G(t), \\ u(t=0) = u_0. \end{cases} \quad (11)$$

Solid part (Nonlinear ODE)

$$\begin{cases} (m + m_a(\delta_G)) \ddot{\delta}_G = -\mathbf{c} \delta_G + \mathbf{c}(\mathbf{e}_1 \cdot u|_{r=R} - h_0) + (\mathbf{b}(u) + \beta(\delta_G)) \dot{\delta}_G^2, \\ \delta_G(0) = \delta_0, \\ \dot{\delta}_G(0) = \delta_1. \end{cases} \quad (12)$$

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Theorem (E.B. '18)

Local well-posedness of the coupled system (11) - (12) for regular enough compatible initial data u_0, δ_0, δ_1

Return to equilibrium

It consists in dropping the solid with no initial velocity from a non-equilibrium position into a fluid initially at rest.

Initial data

Solid: $\delta_G(0) = \delta_0 \neq 0$, $\dot{\delta}_G(0) = 0$

Fluid: $h_e(0, r) \equiv h_0$, $\zeta_e(0, r) \equiv 0$, $q_e(0, r) \equiv 0$

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⇒ Compatibility conditions are **NOT** satisfied

Different approach:

- **linearized** equations in the exterior domain
- **nonlinear** equations in the interior domain

Hydrodynamical linear-nonlinear model (L-NL)

- $r \in (R, +\infty)$

$$\begin{cases} \partial_t \zeta_e + \partial_r q_e + \frac{q_e}{r} = 0 \\ \partial_t q_e + gh_0 \partial_r \zeta_e = 0 \end{cases}$$

- $r \in (0, R)$

$$\begin{cases} \partial_t h_i + \partial_r q_i + \frac{q_i}{r} = 0 \\ \partial_t q_i + \partial_r \left(\frac{q_i^2}{h_i} \right) + \frac{q_i^2}{r h_i} + gh_i \partial_r h_i = -\frac{h_i}{\rho} \partial_r \underline{P}_i \end{cases}$$

- $r = R$

$$q_e|_{r=R} = -\frac{R}{2} \dot{\delta}_G(t), \quad \underline{P}_i|_{r=R} = P_{atm} + \rho g (\zeta_e - \zeta_i)|_{r=R} + P_{cor}$$

Focus on the solid equation

$$(m + m_a(\delta_G))\ddot{\delta}_G(t) = -\mathbf{c}\delta_G(t) + \mathbf{c}\zeta_e(t, R) + \left(\frac{\mathbf{b}}{h_e^2(t, R)} + \beta(\delta_G) \right) \dot{\delta}_G^2(t)$$

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Exterior problem:

$$\begin{cases} \partial_t \zeta_e + \partial_r q_e + \frac{q_e}{r} = 0 \\ \partial_t q_e + v_0^2 \partial_r \zeta_e = 0, \end{cases} \quad \text{B.C.} \quad q_e|_{r=R} = -\frac{R}{2} \dot{\delta}_G(t).$$

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Linear wave equation

$$\partial_{tt} \zeta_e - v_0^2 \Delta_r \zeta_e = 0$$

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Linear wave equation

$$\partial_{tt} \zeta_e - v_0^2 \Delta_r \zeta_e = 0$$

Helmholtz equation with complex coefficients (\mathcal{L} Laplace transform):

$$\begin{cases} s^2 \mathcal{L}(\zeta_e) - v_0^2 \Delta_r \mathcal{L}(\zeta_e) = 0. \\ \partial_r \mathcal{L}(\zeta_e)|_{r=R} = \frac{sR}{2v_0^2} \mathcal{L}(\dot{\delta}_G)(s) \end{cases}$$

Focus on the solid equation

$$(m + m_a(\delta_G))\ddot{\delta}_G(t) = -\mathbf{c}\delta_G(t) + \mathbf{c}\zeta_e(t, R) + \left(\frac{\mathbf{b}}{h_e^2(t, R)} + \beta(\delta_G) \right) \dot{\delta}_G^2(t)$$

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Theorem (E. B. '19)

Considering the linear-nonlinear hydrodynamical model (L-NL), the solid motion is governed by

$$(m + m_a(\delta_G))\ddot{\delta}_G = -\mathbf{c}\delta_G - \nu\dot{\delta}_G + \mathbf{c} \int_0^t F(s)\dot{\delta}_G(t-s)ds + \left(\mathbf{b}(\dot{\delta}_G) + \beta(\delta_G)\right)\dot{\delta}_G^2, \quad (14)$$

The Cauchy problem for (14) with $\delta_0 \neq 0$ and $\delta_1 = 0$ admits a unique solution $\delta_G \in C^2([0, +\infty), \mathbb{R})$ provided some admissibility condition on the initial datum δ_0 . Moreover there exist $M \geq 1, \omega > 0$ and $\epsilon > 0$

$$|\delta_G(t)|^2 + |\dot{\delta}_G(t)|^2 \leq M e^{-\omega t} |\delta_0|^2 \quad \forall t \geq 0 \quad (15)$$

for $|\delta_0| \leq \epsilon$.

Remark

The impulse response function F is (numerically) exponentially decreasing!

Linearizing (14) around the equilibrium state, we get

$$(m + m_a(0)) \ddot{\delta}_G(t) = -c\delta_G(t) - \nu\dot{\delta}_G(t) + c \int_0^t F(t-s)\dot{\delta}_G(s)ds \quad (16)$$

which is a **Cummins**-type equation for the vertical motion.

Cummins equation¹ for the heave

$$(m + a_\infty) \ddot{\delta}_G(t) = -c(\delta_G(t) + z_{G,eq}) - \int_0^t K(\tau)\dot{\delta}_G(t-\tau)d\tau, \quad (17)$$

is implemented in naval architecture and hydrodynamical engineering.

¹Cummins, W.E., *The Impulse Response Function and Ship Motions*, Navy Department, David Taylor Model Basin, 1962.

Numerical results

$h_0 = 15$ m, $R = 10$ m, $H = 10$ m, $\rho = 1000$ kg/m³, $\rho_m = 0.5 \rho$.

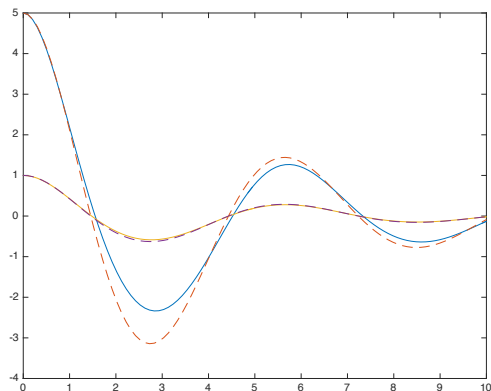


Figure: Time evolution of δ_G given by the nonlinear integro-differential (14) (full) and by the linear Cummins equation (16) (dash) for $\delta_0 = 1$ m and $\delta_0 = 5$ m.

Summary

- We do take into account **nonlinear terms**
- Validation of the shallow water approach to the floating body problem: several experimental data with an **axisymmetric geometry**
- Validation and improvement of the **Cummins** equation

Perspectives

- Add **horizontal** motion and **rotation**: evolution of the contact line + no axisymmetric flow (Iguchi-Lannes '18 in 1d)
- Study the **nonlinear-nonlinear** system for the decay test

THANK YOU FOR THE ATTENTION!